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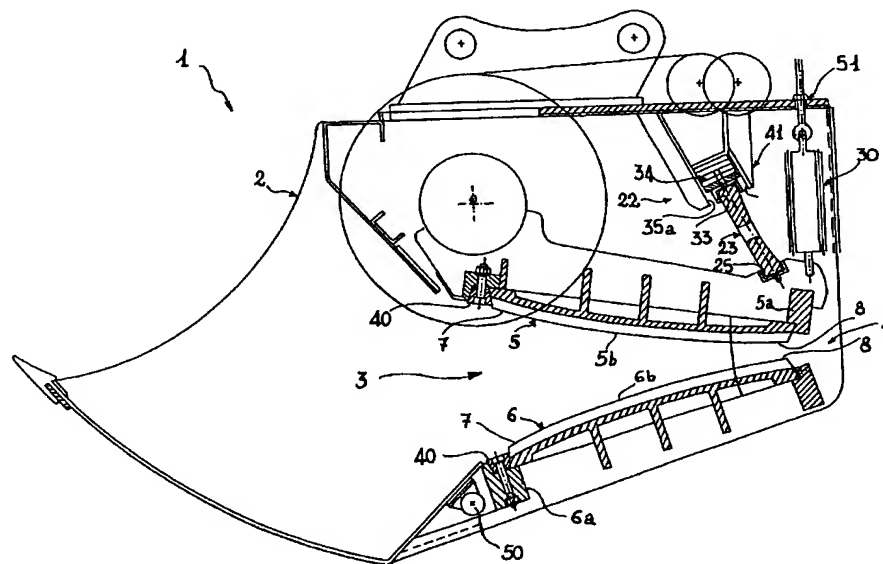
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(54) Title: A BUCKET FOR CRUSHING AND SCREENING STONE



(57) Abstract: A bucket (1) for crushing and screening stone and similar materials is described and comprises a scoop-shaped body (2) defining an inlet opening (3) for the stone to be crushed and an outlet (4) for the crushed stone, between which a direction of flow of the stone is defined, stone-crushing means comprising a first jaw (5) and a second jaw (6) housed in the scoop-shaped body (2) and movable relative to one another, and means for moving the first jaw (5) relative to the second jaw (6) in which a first component of the movement is away from and towards the second jaw (6) and a second component of the movement is substantially parallel to the direction of flow.



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A bucket for crushing and screening stone

Technical field

The present invention relates to a bucket for crushing and screening stone and similar materials, according to the preamble to the main claim.

5 Technological background

In the technical field in question, self-propelled vehicles equipped with buckets for collecting material such as stone or the like, inside which crushing means are provided for crushing the material collected to the desired size, are known.

Amongst others, an example of known crushing means comprises two jaws, of which
10 one moves pivotably relative to the other and which are moved in a manner such as to compress between them, and hence to crush, the material which is introduced into the bucket. However, these crushing means lead to some disadvantages which result in poor performance and non-homogeneity in the processing of the material treated. Known buckets therefore have high power consumption and are subject to blockage
15 due to choking with the material introduced.

Description of the invention

The main object of the present invention is to provide a bucket for crushing and screening stone and similar materials in which the crushing operation is particularly effective and efficient.

20 A further object is to provide a bucket in which the size of the crushed material is easily adjustable.

Another object is to produce a bucket which is subject to little or no obstruction due to blockage with the material treated.

Yet another object is to produce a bucket which can be adapted to a plurality of self-
25 propelled vehicles and which can easily be produced in many different sizes.

A further object is to provide a bucket which permits optimal, in particular homogeneous, crushing of a plurality of different materials.

The objects proposed are achieved by the present invention by means of a bucket formed in accordance with the appended claims.

30 Brief description of the drawings

The characteristics and the advantages of the invention will become clearer from the detailed description of two embodiments thereof, described by way of non-limiting example with reference to the appended drawings, in which:

Figure 1 is a partially-sectioned side view of a bucket according to the invention,

Figure 2 is a partially-sectioned plan view of the bucket of Figure 1,

Figure 3 is a partially-sectioned front view of the bucket of Figure 1,

Figure 4 is a view showing a detail of a further embodiment of the bucket of Figure 1,

5 on an enlarged scale, and

Figure 5 is a view of the detail of Figure 4 in a further operative position.

Preferred embodiments of the invention

In the drawings, a bucket formed in accordance with the present invention is generally indicated 1.

10 The bucket 1 is arranged for connection, in known manner, to one or more arms of a self-propelled vehicle (not shown).

The bucket 1 comprises a scoop-shaped body 2 having an inlet opening 3 for the loading of broken stone, pebbles, stones and the like and having a cross-section which is enlarged in comparison with an opposed outlet opening 4 for the discharge of

15 the material treated, after crushing and screening.

Stone-crushing means are mounted in the scoop-shaped body 2 and comprise a movable crushing jaw 5 and an opposed fixed crushing jaw 6 fixed firmly to the body 2. Both the movable jaw and the fixed jaw 5, 6 include respective frames 5a, 6a on which plates 5b, 6b are fitted removably; the plates 5b, 6b are provided with
20 longitudinal grooves, all indicated 20, extending parallel to the direction of flow of the stone introduced and suitable for facilitating the crushing thereof. The grooves 20 define a plurality of ribs 20a and recesses 20b, alternating in succession in a manner such that a rib 20a of the movable jaw 5 corresponds to a recess 20b of the fixed jaw 6, so that, during the movement of the first jaw 5, the crushing of the material is
25 homogeneous. Moreover, since the ribs 20a of one jaw can penetrate the recesses 20b of the other jaw, the crushing can be particularly fine.

The plates 5b, 6b are reinforced and restrained, by respective undercuts, by means of retaining strips 40.

Respective first and second opposite ends 7, 8 are defined in each of the jaws 5, 6,
30 the first ends 7 of the fixed and movable jaws 6, 5 being positioned in the region of the inlet opening 3, and the second ends 8 being positioned in the region of the outlet 4. The distance between the first ends 7 of the jaws 5 and 6 determines the maximum size of the stone which can be loaded into the bucket and is greater than the distance

between the second ends 8 which, on the other hand, is correlated with the desired maximum size of the crushed stone at the outlet. Both the distance between the first ends 7 and the distance between the second ends 8 are adjustable, as explained in detail below.

- 5 The bucket 1 also comprises means for moving the movable jaw 5, including drive means, for example, a hydraulic motor 9, which is housed inside the scoop-shaped body 2 and drives a drive shaft 10 on which a first pulley 11 is keyed. The rotary movement of the first pulley 11 is transmitted, by means of a belt transmission 12, to a second pulley 13, keyed to a shaft 14.
- 10 A first eccentric 15 and a second eccentric 16 are arranged on the shaft 14, in phase with one another, and each is coupled with a respective first or second bearing 17, 18. A hollow sleeve 19 is fitted on the two bearings 17, 18 so as to be freely rotatable relative thereto and the movable jaw 5 is fixed, in the region of its first end 7, to the outer surface 19a of the sleeve 19, so as to be moved by the shaft 14 together with
- 15 the sleeve 19.
- The drive means 9 are also arranged, when necessary, to drive vibrator means 50 acting on the fixed jaw 6 and disposed in the region of the inlet opening 3, for bringing about pulsed vibrations of the jaw 6 so as to release any material which has become stuck.
- 20 The bucket 1 also comprises means 22 for adjusting the movement of the movable jaw 5 and the size of the cross-section of the outlet 4. The adjustment means 22 comprise a strut 23 interposed and restrained between respective first and second channels 25, 33, of which one is mounted on the frame 5a of the movable jaw 5 and the other on a support 41 fixed firmly to the scoop-shaped body 2. The ends 24a, 24b
- 25 of the strut 23 which are housed in the channels 25, 33 are rounded to facilitate their pivoting about the respective contact lines.
- A set of removable spacers 34 is interposed between the support 41 and the corresponding second channel 33 for the adjustment of the size of the cross-section of the outlet 4. The second channel 33 is welded to the end spacer.
- 30 In a first embodiment of the invention of Figure 1, the channel 33 is welded centrally to the end spacer whereas, in a further embodiment shown in Figures 4 and 5, the channel 33' is welded in the region of an edge of the spacer. By varying the position of the second channel 33, 33' relative to the end spacer, the angle between the

movable jaw 5 and the strut 23 can in turn be adjusted in order to vary in the manner described below. By virtue of the characteristics just described, the strut 23 can be positioned in three different operative positions: a first, central operative position, in which the second channel 33 is spaced equally from two opposed walls 35a, 35b of the support 41, and which can be achieved with the use of the channel 33 welded as shown in Figure 1, a second operative position in which the channel 33' is close to the first wall 35a, and a third position in which it is close to the second wall 35b, which can be achieved, from the second operative position, by removing the spacer and channel 33' and reinserting them having rotated them through 180° (thus changing from the operative position of Figure 4 to that of Figure 5). According to the operative position selected, the angle formed between the strut 23 and the movable jaw 5, in particular, the angle between an axis Y joining the centre of rotation of the pulley 13 and the point P at which the strut 23 is supported in the first channel 25, and an axis Z of the strut 23 extending through the support point P, is varied. This angle is 45°, 40° and 50° in the three operative positions listed above, respectively.

The bucket 1 also comprises resilient means, in particular, a spring 30, a first end of which is connected to the scoop-shaped body 2, and a second, opposite end of which is connected to the second end 8 of the movable jaw 5, so as to keep the strut 23 restrained between the first and second channels 25, 33 (or 33') during the movement of the jaw 5. A mechanism 51 for adjusting the load exerted by the spring 30, such as a screw coupling system, is also provided on the scoop-shaped body 2.

The bucket 1 according to the invention operates as follows.

The stone or other material to be crushed is collected by the bucket 1 in conventional manner. In order to send the material collected towards the jaws 5, 6, the bucket 1 is pivoted through 90° from the position in which it is shown in Figure 1, that is, the outlet 4 is arranged at a height below the inlet opening 3 so that the material is urged towards the jaws 5, 6 simply by the effect of gravity.

The flow of material is facilitated with the use of the vibrator means 50, even if the inlet opening 3 is positioned at the same height as the outlet 4.

The movable jaw 5 is moved by operation of the hydraulic motor 9 which transfers the movement from the first pulley 11 to the second pulley 13 and consequently to the shaft 14. Owing to the effect of the two eccentrics 15, 16, the sleeve 19, which is freely rotatable on the bearings 17, 18, can perform a rotational/translational

movement relative to the axis of the shaft 14; in particular, the first end 7 of the movable jaw 5, which is fixed to the sleeve 19, is moved from a first position, in which the inlet opening 3 has a maximum cross-section, to a second, opposite position which differs from the first by a rotation of the eccentrics 15, 16 through 180° , and in which the inlet opening 3 has a minimum cross-section. The first end 7 of the movable jaw 5 adopts all of the intermediate positions between the above-defined first and second positions, during its rotational/translational movement.

Since the movable jaw 5 is a rigid body, movements of the first end 7 result in corresponding movements of the second end 8 which, however, is restrained by the spring 30 and by the strut 23. The movements of the end 8 are permitted by the pivoting of the ends 24a, 24b of the strut 23 within the first and second channels 25, 33 (33'), respectively, so that the inclination of the strut 23 relative to the jaw 5 is varied continuously during the movement of the jaw 5. The resulting movement comprises a component substantially perpendicular to the jaw 5 and a component parallel thereto, along the direction of flow of the stone, in a manner similar to a "chewing" motion, promoting crushing of the stone and its movement towards the outlet 4.

The maximum size of the cross-section of the outlet 4 can also be adjusted by increasing or reducing the number of spacers 34 located inside the support 41, thus varying the maximum size of the crushed stone.

The movement of the jaw 5 can also be modified, thus changing the characteristics of the crushing due to the relative movement of the jaws 5, 6, by varying the inclination between the strut 23 and the jaw 5 at rest, as described above. With the use of the configuration shown in Figure 4, in which the inclination between the strut 23 and the jaw 5 is least, the movement of the end 8 of the movable jaw 5 comprises a considerable translational component in the direction of the flow of the material, thus facilitating the movement of the material towards the outlet 4. This position is therefore particularly suitable when materials which form blockages easily, for example, moist or fine-grained materials, are being processed. The positioning shown in Figure 5, which can be achieved by rotating the channel 33' welded to the spacer through 180° , on the other hand, is particularly suitable when a considerable crushing power is required.

The invention thus achieves the objects proposed, also affording numerous advantages over the prior art referred to.

A first advantage afforded by the bucket according to the invention is that it is possible to optimize the crushing of the stone by virtue of the plurality of adjustments permitted,

5 by adjusting the relative movement of the jaws in dependence on the material.

Moreover, the size of the crushed stone can easily be adjusted.

One of the main advantages is that the power consumption of the bucket according to the invention is less than that of conventional buckets, by virtue of the greater efficiency achieved by the process, which also leads to a reduction in processing time

10 and to a reduction in noise emitted.

Moreover, the crushing performed by means of the above-described bucket is particularly uniform.

One of the main advantages is that, by virtue of the type of movement of the jaw and of the provision of vibrator means, blockages of material and consequent stoppages of

15 the processing are minimized.

Claims

1. A bucket (1) for crushing and screening stone and similar materials, comprising a scoop-shaped body (2) defining an inlet opening (3) for the stone to be crushed and an outlet (4) for the crushed stone, between which a direction of flow of the stone is defined, means for crushing the stone, the crushing means comprising a first jaw (5) and a second jaw (6) housed in the scoop-shaped body (2) and movable relative to one another, and means for moving the first jaw (5) relative to the second jaw (6), characterized in that the movement means can impart to the first jaw (5) a combined rotational and translational movement relative to the second jaw (6), in which a first component of the movement is away from and towards the second jaw (6) and a second component of the movement is substantially parallel to the direction of flow.
2. A bucket (1) according to Claim 1, comprising means (22) for adjusting the size of the cross-section of the outlet (4) and the movement of the first jaw (5).
3. A bucket (1) according to Claim 1 or Claim 2 in which the first jaw (5) and the second jaw (6) comprise respective first and second opposite ends (7, 8) which are positioned, with reference to the direction of flow, in the region of the inlet opening (3) and in the region of the outlet (4), respectively, the movement means acting on the first end (7) of the first jaw (5).
4. A bucket (1) according to Claim 3 in which the second end (8) of the first jaw (5) is coupled with means (22) for adjusting the size of the cross-section of the outlet (4) and the movement of the first jaw (5).
5. A bucket (1) according to any one of Claims 3 and 4 in which the movement means comprise at least one eccentric (15; 16) on which a sleeve (19) is coupled in a freely rotatable manner, the sleeve (19) being fixed firmly to the first end (7) of the first jaw (5).
6. A bucket (1) according to Claim 5 in which the movement means comprise two eccentrics (15, 16) moved by a shaft (14) driven by drive means (9), the two eccentrics (15, 16) being coupled with two bearings (17, 18) on which the sleeve (19) is fitted.
7. A bucket (1) according to any one of Claims 3 to 6 in which the adjustment means (22) comprise a strut (23) interposed at an adjustable inclination between the second end (8) of the first jaw (5) and the scoop-shaped body (2).

8. A bucket (1) according to Claim 7 in which the second end (8) of the first jaw (5) comprises a first channel (25) for housing a first end (24a) of the strut (23) in an orientable manner.
9. A bucket (1) according to Claim 8 in which a support (41) is mounted on the scoop-
5 shaped body (2) and houses a set of removable spacers (34), the spacers (34) being interposed between the support (41) and a second channel (33; 33') which houses a second end (24b) of the strut (23).
10. A bucket (1) according to Claim 9 in which the second end (24b) of the strut (23) is housed in the second channel (33; 33') in alternative operative positions, in order to
10 adjust the inclination between the strut (23) and the first jaw (5), at rest.
11. A bucket (1) according to one or more of the preceding claims in which each of the first jaw (5) and the second jaw (6) comprises a respective frame (5a, 6a) on which respective plates (5b, 6b) are fitted removably.
12. A bucket (1) according to Claim 11 in which a plurality of grooves (20) are formed
15 on facing surfaces of the plates (5b, 6b).
13. A bucket (1) according to Claim 12 in which the grooves (20) are parallel to one another and extend in the direction of the flow of the stone.
14. A bucket (1) according to Claim 13 in which the grooves (20) define a plurality of ribs (20a) and recesses (20b) alternating in succession in a manner such that a rib
20 (20a) of the first jaw (5) corresponds to a recess (20b) of the second jaw (6).
15. A bucket (1) according to one or more of the preceding claims in which the second jaw (6) is fixed firmly to the scoop-shaped body (2).
16. A bucket (1) according to one or more of Claims 7 to 15, comprising resilient means (30) suitable for resiliently urging the second end (8) of the first jaw (5) against
25 the strut (23).
17. A bucket (1) according to Claim 16, comprising means (51) for adjusting the load of the resilient means (30).
18. A bucket (1) according to one or more of the preceding claims, comprising vibrator means (50) disposed in the region of the inlet opening (3) for bringing about pulsed
30 vibration of the second jaw (6).

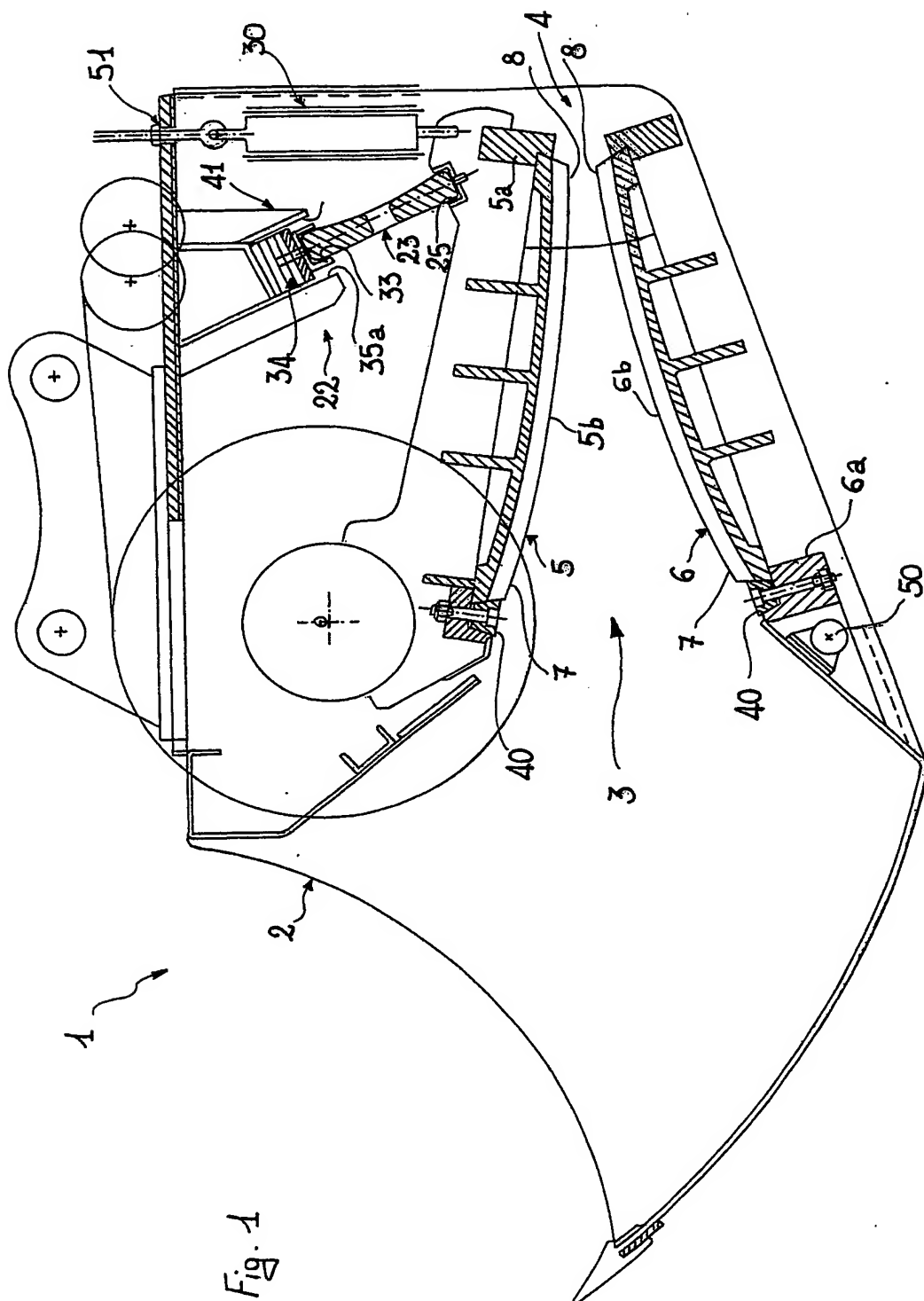
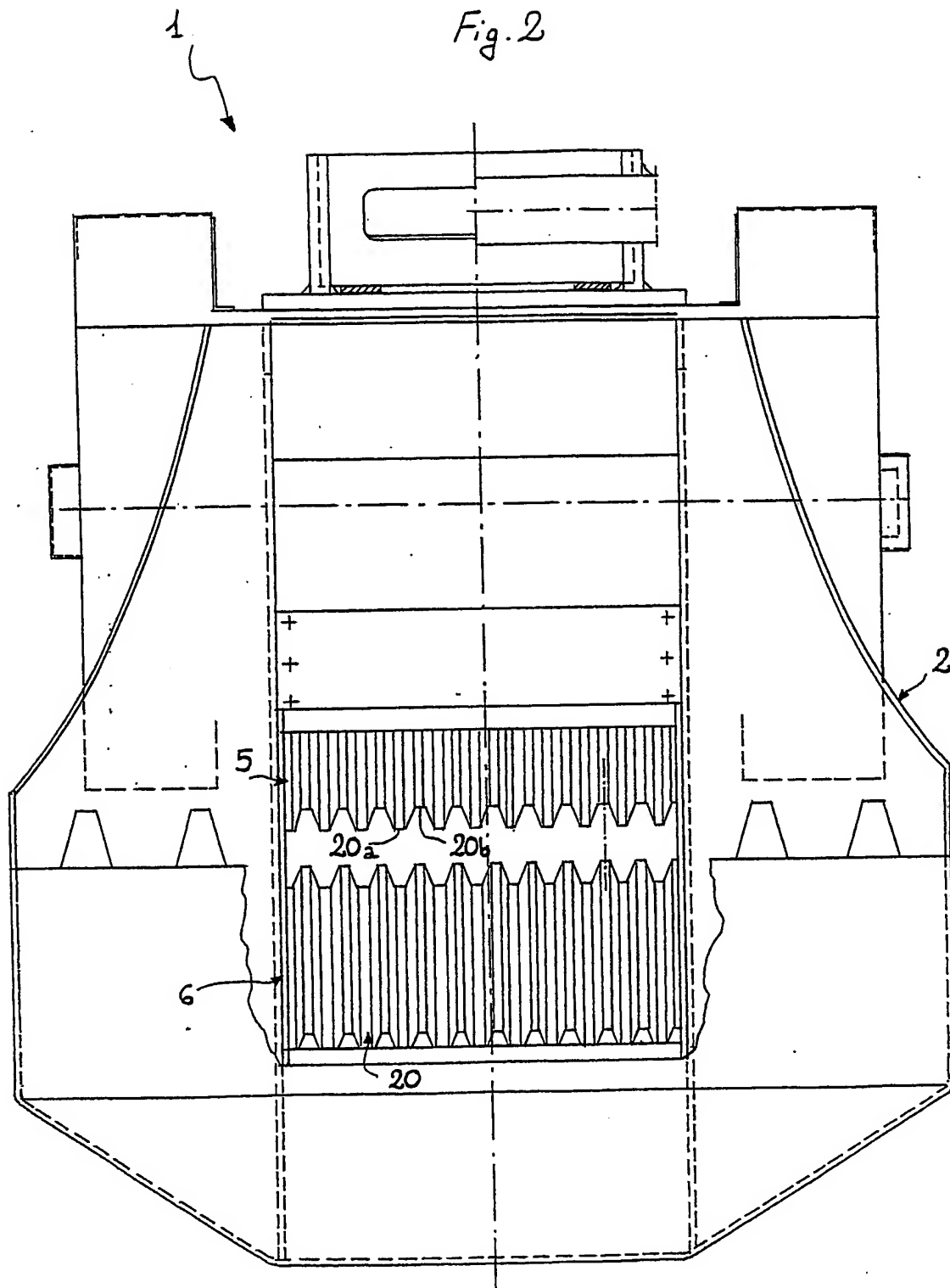
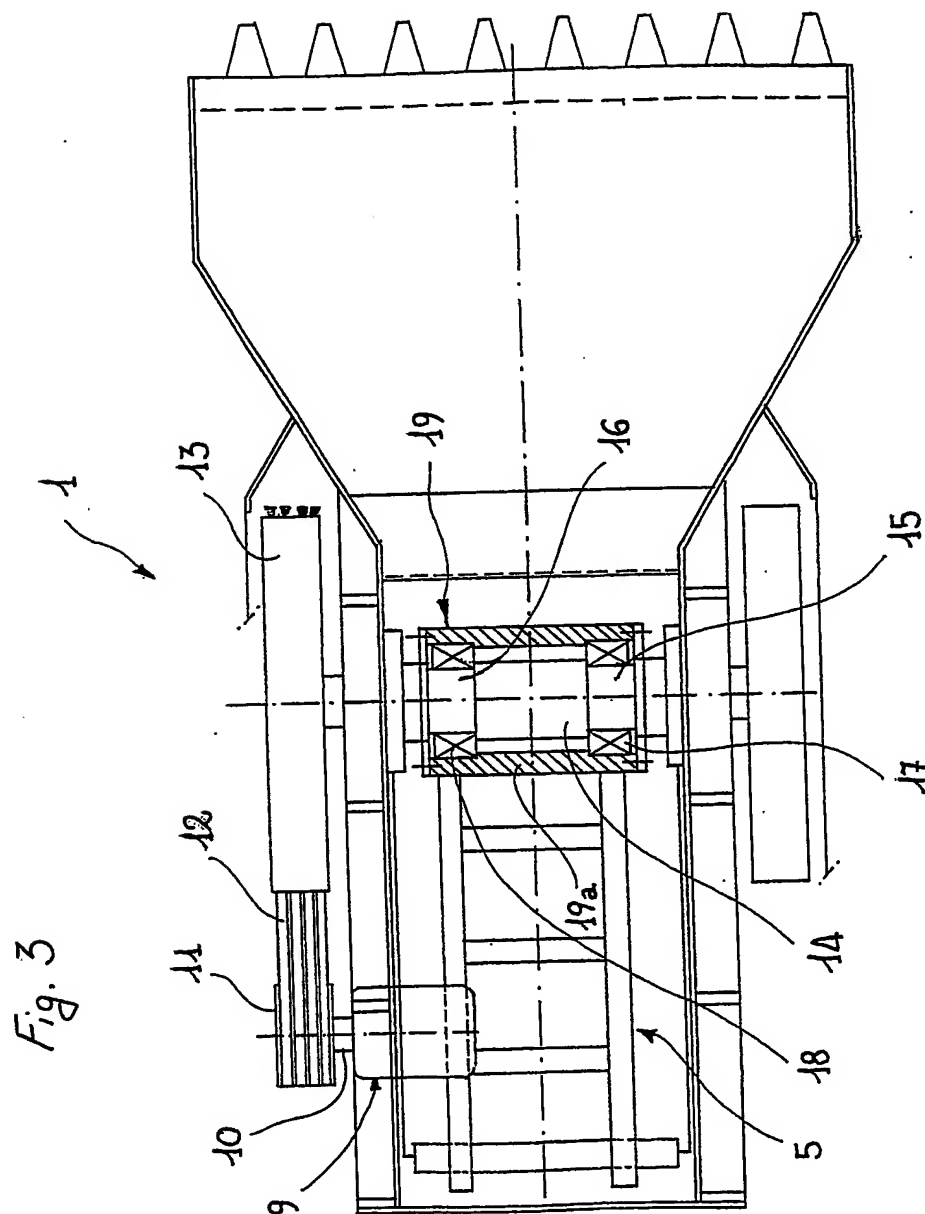


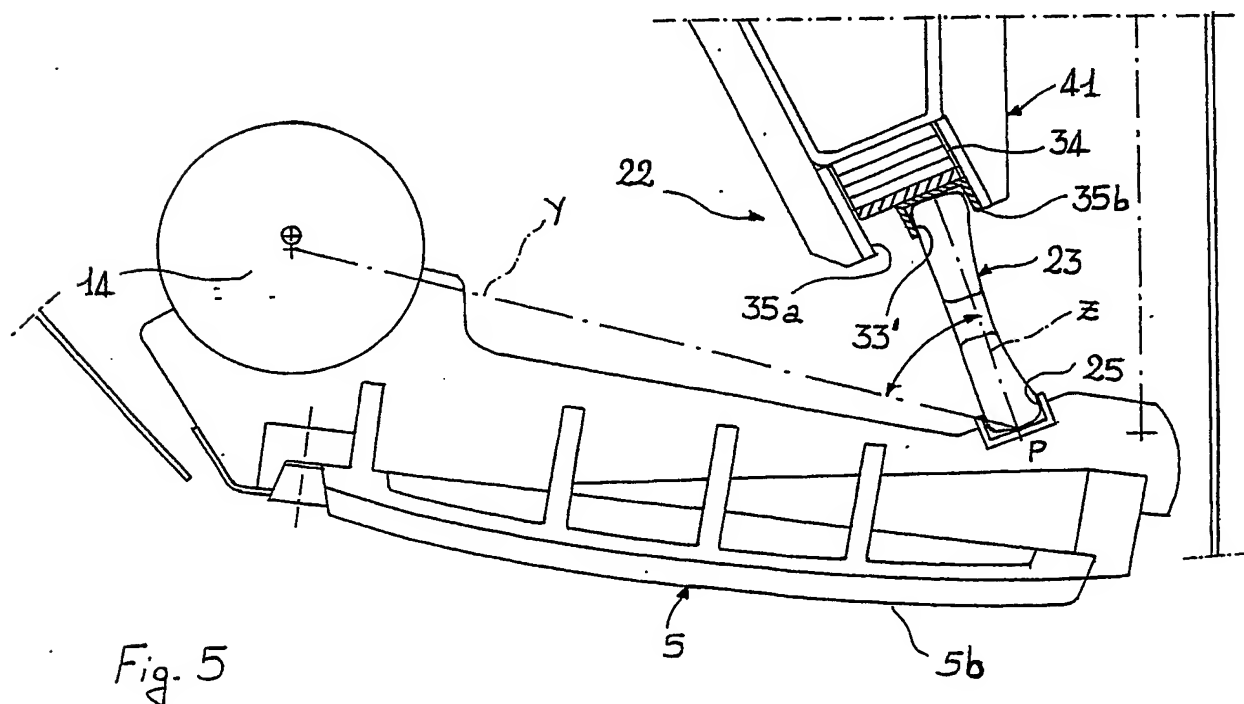
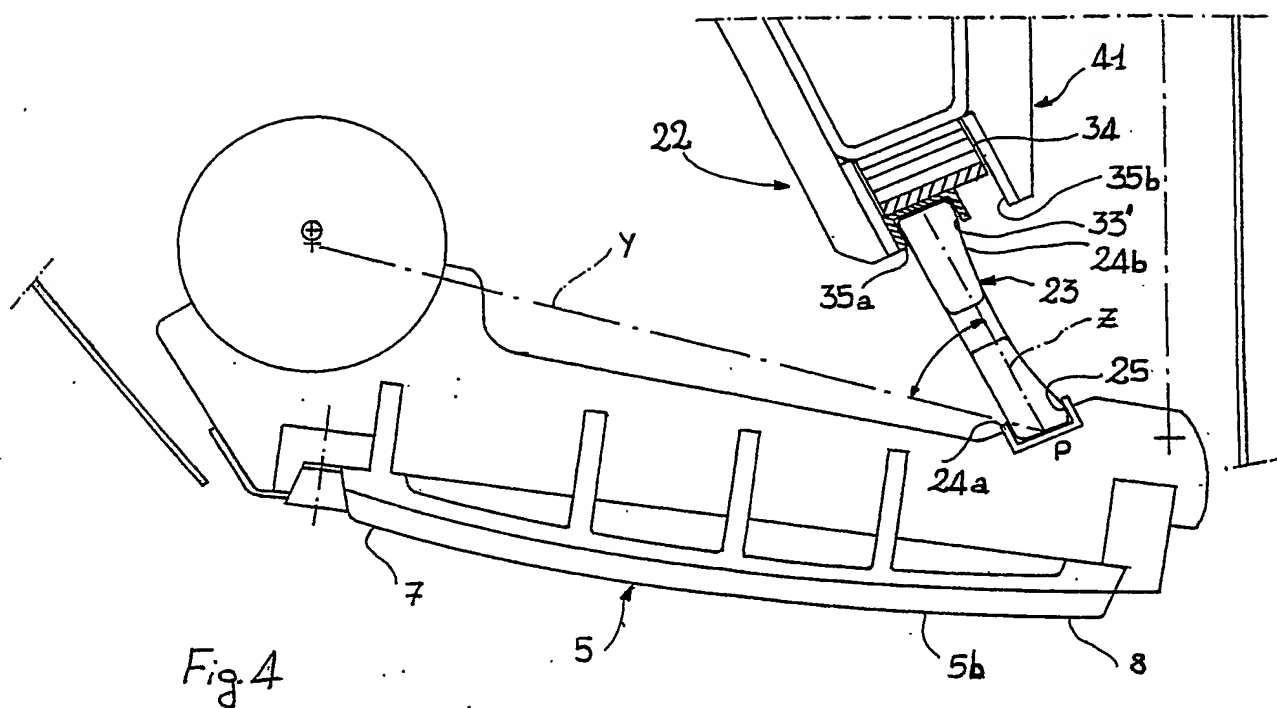
Fig. 1

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Fig. 2







INTERNATIONAL SEARCH REPORT

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According to International Patent Classification (IPC) or to both national classification and IPC

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Minimum documentation searched (classification system followed by classification symbols)
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Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)

EPO-Internal

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category *	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
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Y	column 1, line 54 - line 65; figures 2,3 column 3	6-10,14
Y	US 1 954 288 A (FRANCIS PAUL R) 10 April 1934 (1934-04-10) column 1, line 34 -column 2, line 102; figures 1,2	6-10
Y	DE 580 475 C (ARTHUR WIENBEIN) 12 July 1933 (1933-07-12) figures 2,3	14
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☒ Patent family members are listed in annex.

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INTERNATIONAL SEARCH REPORT

International Application No
PCT 02/00555

C.(Continuation) DOCUMENTS CONSIDERED TO BE RELEVANT

Category *	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
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INTERNATIONAL SEARCH REPORT

Information on patent family members

International Application No

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